



QUALITATIVE STUDY OF THE STRUCTURAL INFLUENCE OF THE FORCES ON THE STABILITY OF DYNAMICAL SYSTEMS

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Abstract: In this paper consider the autonomous dynamical system linear or linearized with 2 degree of freedom. In the system of equation of 4th degree, appear the structure generalized forces: $\mathbf{K}(\mathbf{q})$ - the conservative forces, $\mathbf{N}(\mathbf{q})$ - the non-conservative forces, $\mathbf{D}(\dot{\mathbf{q}})$ the dissipative forces, $\mathbf{G}(\dot{\mathbf{q}})$ the gyroscopically forces. In the linear system, these forces from the different structural combinations can produce the stability or the instability of the null solution. In this way are known the theorems of Thomson - Tait - Cetaev (T-T-C) for the configurations $(\mathbf{K}, \mathbf{D}, \mathbf{G})$. We will introduce the non conservative forces \mathbf{N} , studying the stability with the Routh - Hurwitz criterion or construct the Liapunov function, obtaining some theorems with practical applications.

Keywords: qualitative theory, stability, system structures, decomposition.

1. INTRODUCTION

In this section study the structural influence of the terms blocks on the stability of the null solution for the bi-dimensional system or equations with fourth degree, which is a linear or linearized system in first approximation for the nonlinear system [2], [4].

$$\begin{cases} \ddot{x}_1 + k_{11}x_1 + k_{12}x_2 + c_{11}\dot{x}_1 + c_{12}\dot{x}_2 + g_{11}\dot{x}_1 + g_{12}\dot{x}_2 + n_{11}x_1 + n_{12}x_2 = 0 \\ \ddot{x}_2 + k_{21}x_1 + k_{22}x_2 + c_{21}\dot{x}_1 + c_{22}\dot{x}_2 + g_{21}\dot{x}_1 + g_{22}\dot{x}_2 + n_{21}x_1 + n_{22}x_2 = 0 \end{cases} \quad (1)$$

In this system the matrix blocks $\mathbf{Kx}, \mathbf{C}\dot{\mathbf{x}}, \mathbf{G}\dot{\mathbf{x}}, \mathbf{Nx}$ are representing respectively the conservative (elastic) forces, the resistance (amortization) forces, the gyroscopically forces and the non-conservative forces. The characteristic polynomial for the Routh - Hurwitz criterion will be [1], [7]:

$$P(\lambda) = \begin{vmatrix} \lambda^2 + (c_{11} + g_{11})\lambda + k_{11} + n_{11} & (c_{12} + g_{12})\lambda + k_{12} + n_{12} \\ (c_{21} + g_{21})\lambda + k_{21} + n_{21} & \lambda^2 + (c_{22} + g_{22})\lambda + k_{22} + n_{22} \end{vmatrix} = 0 \quad (2)$$

The system (1) can be bring to the canonic form and making abstraction by the constant negative factor the stated forces will be respectively side by the system (x_1, x_2) :

$$\begin{aligned} & \bar{\mathbf{F}}(k_{11}x_1 + k_{12}x_2, k_{21}x_1 + k_{22}x_2), \\ & \bar{\mathbf{C}}(c_{11}\dot{x}_1 + c_{12}\dot{x}_2, c_{21}\dot{x}_1 + c_{22}\dot{x}_2), \\ & \bar{\mathbf{G}}(g_{11}\dot{x}_1 + g_{12}\dot{x}_2, g_{21}\dot{x}_1 + g_{22}\dot{x}_2), \\ & \bar{\mathbf{N}}(n_{11}x_1 + n_{12}x_2, n_{21}x_1 + n_{22}x_2). \end{aligned}$$

Regarding this system there are classical contributions of the Liapunov and the theorems of Thomson - Tait - Cetaev [4], Merkin [2] and Crandall [4]. Here we'll distinguish these results and we'll make other structural contribution by examples.

In the matricial calculus are known the decomposition theorems of the squared matrix. Any squared matrix can be decompose in a sum of one symmetric matrix and the other one asymmetric $\mathbf{M} = \mathbf{A} + \mathbf{B}$. where $\mathbf{A} = \frac{1}{2}(\mathbf{M} + \mathbf{M}')$, $\mathbf{B} = \frac{1}{2}(\mathbf{M} - \mathbf{M}')$.

From the decomposition theorem and using the fact that the positional forces $\bar{\mathbf{K}}$ are conservative with

$rot\bar{K} = 0$, $\bar{K} = -gradU(q)$ and $rot\bar{N} \neq 0$ and $rot_{\dot{q}}\bar{C} = 0$, $\bar{C} = -grad_{\dot{q}}V(q)$, $rot\bar{G} \neq 0$ we have the condition of symmetry and asymmetry $k_{ij} = k_{ji}$, $n_{ij} = -n_{ji}$, $g_{ij} = -g_{ji}$, $c_{ij} = c_{ji}$, $i, j = 1, 2$.

We have the relations:

$$\begin{cases} \ddot{x} + c_1\dot{x} + g\dot{y} + k_1x - py = X^s(0) \\ \ddot{y} + c_2\dot{y} - g\dot{x} + k_2y + px = Y^s(0) \end{cases} \quad (3)$$

The system (3) has the fourth degree, and the characteristic polynomial is:

$$P(\lambda) = det \begin{vmatrix} \lambda^2 + \lambda c_1 + k_1g\lambda - p & \\ -g\lambda + p\lambda^2 + c_2\lambda + k_2 & \end{vmatrix} = 0 \quad (4)$$

$$P(\lambda) = \lambda^4 + \lambda^3(c_1 + c_2) + \lambda^2(k_1 + k_2 + c_1c_2 + g^2) + \lambda(c_1k_2 + c_2k_1 + 2gp) + k_1k_2 + n^2 = 0 \quad (5)$$

The system with constant coefficients (1) becomes:

$$\begin{cases} \ddot{x} + ax + ky + p\dot{x} + c\dot{y} + g\dot{y} - ny = 0 \\ \ddot{y} + kx + by + c\dot{x} + q\dot{y} - g\dot{x} + nx = 0 \end{cases} \quad (6)$$

The mechanical justification of this configuration is obtain starting from the Lagrange equations:

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}} \right) - \frac{\partial T}{\partial q} = Q_1(q) + Q_2(\dot{q}) + Q_s(q, \dot{q}) \quad (7)$$

where T is the kinetic energy, Q_1, Q_2 are the generalized force and Q_s are generalized forces with superior order terms. So, the decomposition mentioned become symbolic for the positional forces and the inertial forces.

$$\bar{Q}_1(q) = \bar{F}(q) + \bar{N}(q); \bar{Q}_2(\dot{q}) = \bar{C}(\dot{q}) + \bar{G}(\dot{q}) \quad (8)$$

where $\bar{F} = -gradU(q)$, $\bar{C}(\dot{q}) = -grad_{\dot{q}}V(q)$, with $U(q)$ the potential energy, $U(\dot{q}) - \sum \sum c_{ij} \dot{q}_i \dot{q}_j$ is the function of dissipation with mechanical work negative, $\bar{N}(q)$ the conservative forces, $\bar{G} = \sum \sum g_{ij} \dot{q}_i \dot{q}_j$, $g_{ij} = -g_{ji}$ with mechanical work null. The system is completely dissipative if $U(\dot{q})$ is the squared form positive defined, and $\frac{d}{dt}(T + U) = -2H$.

Theorem 1. If $\bar{Q}_1 = -gradU(q) + N(q)$ then the expression of the Q_1 block is: $Q_1 = K(q) + N(q)$.

Theorem 2. If $\bar{Q}_2(\dot{q}) = -gradH(\dot{q}) + \bar{D}(\dot{q})$ then the expression of the Q_2 block is: $Q_2 = G(\dot{q}) + D(\dot{q})$

On the base of this structure of forces $K(q), N(q), D(\dot{q}), G(\dot{q})$ we'll analyze the stability, making some combinations of these forces, and we'll obtain a series of theorems. For the system K, D, G , Thomson - Tait - Cetaev (T-T-C) has defined the theorems.

The examples are for the gyroscopes, bearings on the fluid support, double pendulum, electron in the magnetic field, the car equation and other examples from different domains making analogies for the system of the fourth degree.

Will note in the next theorem with Σ the system (ex. $\Sigma(K, D)$ - the system compose of K and D), with S the stable case, with $A.S$ the asymptotic case, and with I the unstable case, using directly the characteristic polynomial (5) (H-R) or the Liapunov function for (6).

2. THE STUDY OF THE STABILITY OF THE DYNAMICAL SYSTEMS

For start we will consider the stability of the equilibrium position in the point of minimum of the potential energy (the theorem Lagrange-Dirichlet) with slight oscillations around this position. We note with $K_0(q)$ the case of the cyclical coordinates in the phases plane when is obtain an uniform movement in report with these. Using the equations of Routh - Hurwitz - the case Lagrange - Poisson, for the solid with a fixed point (the gyroscope) is studied the stability of the uniform movements.

Theorem 3. If $-\frac{\partial U}{\partial q} = K_0(q)$ the dynamical system $\Sigma(K_0(q))$ is dynamical stable around of $q_0 = 0$. (Ex.: the mathematical pendulum in the graviphic field)

$$S(K_0) \Rightarrow S(\sum K_0)$$

Observation 1. If K_0 is stable then $S(\sum K_0, G)$ is stable.

Observation 2. If K_0 is non stable then $S(\sum K_0, G)$ is non stable.

Theorem 4. (T-T-C) In the conservative system, if potential energy has an isolated minimum, then the system is simple stable around the minimal point: if K_0 is stable $S(\sum K_0) \Rightarrow S(\sum(K_0 + (G, D)))$ around the zero point.

Theorem 5. (T-T-C) If we have an isolated potential simple stable then by attach to the system the dissipative forces the the simple stability is keep; if the dissipation is total then the system become asymptotic stable: if K_0 is stable $S(\Sigma K_0) \Rightarrow sum(K_0 | (G, D_c)) A.S$, D_c is complete dissipative, and in this case the system is asymptotic stable.

Theorem 6. (T-T-C) In an instable potential regime in the vicinity of a maximum point for the potential energy, when this energy is negative, if are applied the dissipative force the instability is keep: if K_0 is unstable then $\Sigma(K_0 + G + D_c)$ is unstable: $I(K_0) \Rightarrow I(\Sigma(K_0 + G + D_c))$

The theorems (3-6) are verified directly with the Routh - Hurwitz criterion for the system (13).

Theorem 7. If G is stable then we have an uniform movement stable of G (stability about \dot{q}). Ex.: the case Lagrange - Poisson, the Routh method when we have the rotation angle and the precession angle cyclically implies the uniform movement.

Theorem 8. If $det G \neq 0$ the stability is conserve compared with the coordinates and with the speeds.

Corollary 1. If in the time of the stable movement $det G = 0$ the stability will be lost just compared with the coordinates, but not compared with the speeds.

Corollary 2. If the system is nonlinear under the acting on G and stable with $det G \neq 0$ then is not implicate the stability for the non linear system.

Theorem 9. (T-T-C) If the system $\Sigma(G + D_c) A.S$. is conserve for the coordinates and speeds and for the non linear system.

Theorem 10. If we are acting with non-potential forces then the system is unstable $\Sigma(N) \Rightarrow I(\Sigma(N))$. (see the Application 1)

Theorem 11. If we have the system $\Sigma(N + D_c)$ then the system is unstable.

Theorem 12.

1. For the system $(K + N)$ we make the next hypothesis: if the system is stable then $\Sigma(K + N)$ is perturbed (can be stable and unstable).

2. If the system $(K + G)$ is stable or unstable then the system $(K + G + N)$ is perturbed (can be stable and unstable).

Theorem 13. The dissipative forces can influenced on the stability $\Sigma(K + N)$ so: if $\Sigma(K | N)$ is unstable then $\Sigma(D + K + N)$ can be stabilized; if $\Sigma(K + N)$ is stable then $\Sigma(K + D + N)$ can be destabilized.

Theorem 14. If in the system K the two equations of second degree has the frequencies equal and by introducing N for the system $\Sigma(K + N)$ we have: if the system $\Sigma(K + N)$ are linear then the stability is perturbed indifferent of the nonlinear terms.

Theorem 15. If $(G + N)$ is unstable then $\Sigma(G + N + D)$ is stable.

It is observe that the theorem T-T-C do not conserve always if appear the non-potential forces (N) from Merkin [2].

Application1. (Theorem 10)

A system which has just the non-potential forces is always unstable. Such a system is represented by equations:

$$\begin{aligned} \ddot{x} + py &= 0 \\ \ddot{y} - px &= 0 \end{aligned} \tag{9}$$

The characteristic equation is $\lambda^4 + p^2 = 0$, with solutions $\lambda = \pm\sqrt{2}/2(1 \pm i)p$ having the roots with the real part positive, so we have the instability of the system.

Application 2. (Theorem 12)

In the case of the systems action by the conservative and non-potential forces we have the equation:

$$\begin{aligned} \ddot{x} + k_1x + py &= 0 \\ \ddot{y} + k_2y - px &= 0 \end{aligned} \tag{10}$$

The characteristic equation $\lambda^4 + (k_1 | k_2)\lambda^2 + k_1k_2 | p^2 = 0$ must have the real and negative roots in λ^2 . The stability domains are presented in the figure below:

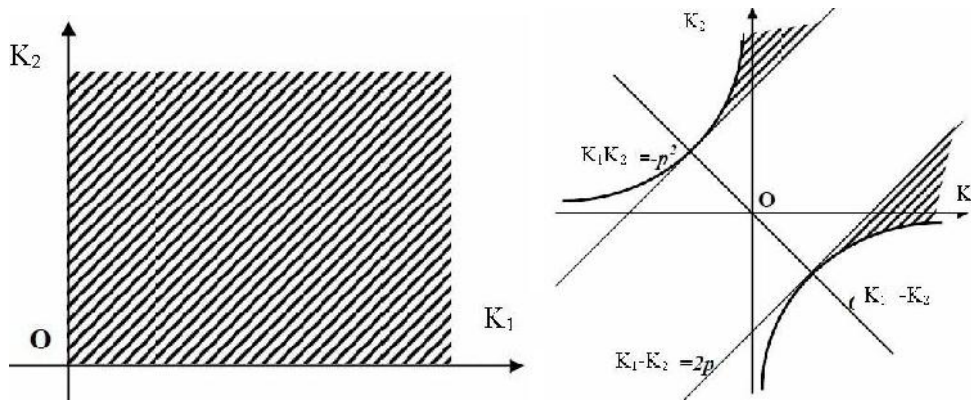


Figure 1: The domain of stability

Application 3. (Theorem 13)

For the system action by the conservative and non potential forces, introducing the dissipative forces we have (fig. 1):

$$\begin{cases} \ddot{x} + c_1\dot{x} + k_1x + py = 0 \\ \ddot{y} + c_2\dot{y} + k_2y - px = 0 \end{cases} \quad (11)$$

The characteristic equation is:

$$\lambda^4 + (c_1 + c_2)\lambda^3 + (k_1 + k_2 + c_1c_2)\lambda^2 + (k_1c_2 + k_2c_1)\lambda + k_1k_2 + p^2 = 0$$

Application 4. The gyroscopic pendulum with 2 degree of freedom x, y and 2 types of amortization [1]:

one linear stationary amortization $(c_s\dot{x}, c_s\dot{y})$ and one rotational amortization $(c_r(\dot{x} + \omega y), c_r(\dot{y} - \omega x))$ with the equations:

$$\begin{cases} I_0\ddot{x} + (c_s + c_r)\dot{x} + J\omega\dot{y} + c_r\omega y - ax\delta = 0 \\ I_0\ddot{y} + (b_s + b_r)\dot{y} - J\omega\dot{x} + c_r\omega x - ay\delta = 0 \end{cases}$$

with I_0 the polar inertial momentum, J the axial inertial momentum, ω the rotation speed, $\delta = \pm 1$. We observe that the conservative forces are: $F(-a\delta x, a\delta y)$, the amortization forces $(b_s + b_r)\dot{x}, (b_s + b_r)\dot{y}, G(J\omega\dot{y}, -J\omega\dot{x}), N(c_r\omega y, c_r\omega x)$.

From the theorem (Thomson - Tait - Cetaev)[3] the movement will be unstable being compose by a stable movement and the other one unstable.

The example of Crandall [3] shows that at the fast speed the movement is stable by using these amortizations, depend of the critical coefficient $r_0 \frac{c_s}{c_r}$ obtain a critical speed of amortization $\omega_c = \omega_p(1 + r)$. On the other

side we observe the presence of the forces N which introduce the unstable zones and stable zone from Merkin which do not conserve the theorem (T-T-C).

Application 5. The cylindrical bearing with the rotor in the viscous fluid, with the center $C(x, y)$, [8]:

The stability of the rotor centre (K, G, N) [7]:

$$\begin{cases} \ddot{x} + b\dot{x} + \omega^2x - py = X \\ \ddot{y} + b\dot{y} + \omega^2y + px = Y \end{cases} \quad (12)$$

Here we have the forces: $K(\omega^2x, \omega^2y), G(b, b), N(-p, p)$, where N represents the aerodynamically forces produced by the rotor in the viscous fluid; the characteristic polynomial is:

$$P(\lambda) = \lambda^4 + 2b\lambda^3 + (2k^2 + b^2)\lambda^2 + 2bk^2\lambda + p^2 + k^4 = 0 \quad (13)$$

If $p = 0$ then the stability domain is in the first quadrant. If $p \neq 0$ then the stability is disappear, so the non potential forces ($p \neq 0$) can make the stability or can extend the stability (outside the first quadrant) (fig.1) [2].

Application 6. The gyroscope with two plans [1], [2]

There is keep the stability by the horizontal plan with α angle and by the vertical plan with β angle.

$$\begin{cases} J\ddot{\alpha} + b\dot{\alpha} - H\dot{\beta} - p\beta = 0; (DGN) \\ J\ddot{\beta} + b\dot{\beta} + H\dot{\alpha} + p\alpha = 0; D(b, b); G(H, -H); N(p, -p) \end{cases} \quad (14)$$

Using the Hurwitz criterion of stability we have:

$$\begin{cases} \Delta_3 = a_1a_2a_3 - a_0^2a_3^2 - a_1^2a_4 > 0; \Delta_2 > 0, \Delta_1 > 0, \alpha, \beta, p, H > 0 \\ \Delta_3 = 4pJ(H^2 + b^2)(bH - p) > 0 \end{cases}$$

For $b = 0$ we have instability $\Delta_3 < 0$ and for $b \neq 0, b > \frac{pJ}{H}$ the system is asymptotic stable.

Application 7. The double pendulum with elastic articulations and a non-conservative force (K, N) [3], [4], [5].

The governing equation are:

$$\begin{cases} a_{11}\ddot{\varphi}_1 + a_{12}\ddot{\varphi}_2 + l_1\varphi_1 - l_2\varphi_2 = 0 \\ a_{21}\ddot{\varphi}_1 + a_{22}\ddot{\varphi}_2 - c_1\varphi_1 - c_2\varphi_2 = 0 \end{cases} \quad (15)$$

For the asymptotic stability we have for the polynomial characteristic $a\lambda^4 + b\lambda^2 + c = 0$ the conditions: $b > 0, \delta = b^2 - 4ac > 0$

Application 8. The automobile with automatic decompression [4]

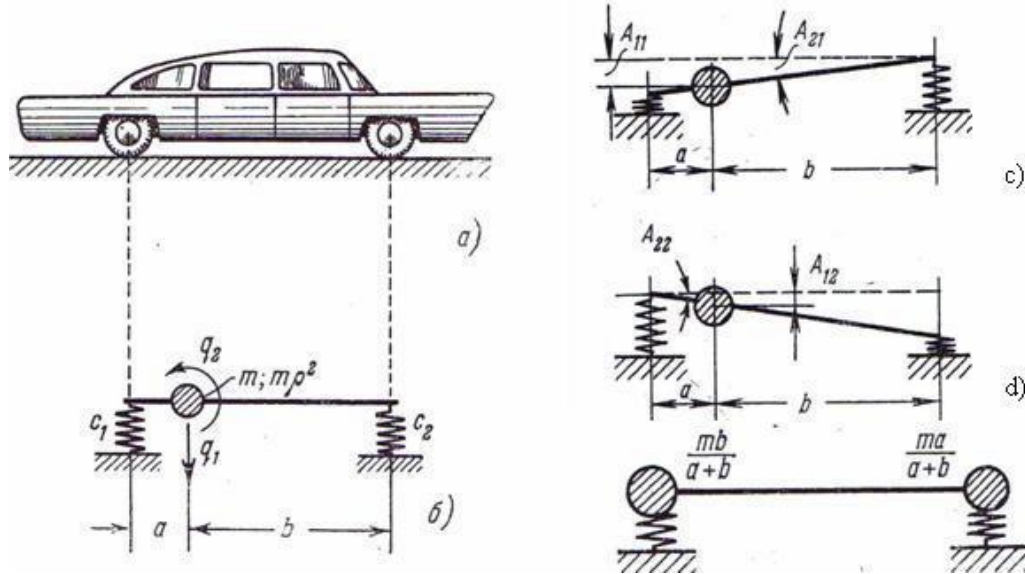


Figure 2: The stability of the automobile

The governing equations are:

$$\begin{cases} m\ddot{x} + (k_1 + k_2)x + (k_1a - k_2b)y = 0 \\ m\rho^2\ddot{y} + (k_1a - k_2b)x + (k_1a^2 + k_2b^2)y = 0 \end{cases} \quad (16)$$

where ρ is the inertial radius. For make the decompression of the two equations, to have a noiseless automobile it must: $k_1a - k_2b = 0$. This thing implies that ρ (the gyration center) $\rho^2 = \sqrt{ab}$ (fig. 2). In the figure 2c and 2d are decompose the movements in caper and gallop.

Finally we present the indeterminate coefficients method, building Liapunov function for systems of fourth degree. The characteristic polynomial is:

$$\lambda^4 + d\lambda^3 + c\lambda^2 + b\lambda + a = 0 \quad (17)$$

The H-R criterion for the S.A of the null solutions: $d > 0, c > 0, a > 0, bcd - b^2 - d^2 > 0$.

Next we'll construct the Liapunov function (V) for the system of 4th degree. There are finding some functions with four variables under the quadratic form:

$$V = \sum_{i,j=1}^4 V_{ij} x_i y_j, W = \sum_{i,j=1}^4 W_{ij} x_i y_j, \dot{V} = 2W \quad (18)$$

where V_{ij} are unknown, W_{ij} are known; using the methods of undetermined coefficients, is derived V for the attach system having four equations with the unknowns V_{ij} . In the algebraic linear system we have D the determinant, and we can choose $W = 2Dy^2$ and the identification $\dot{V} = 2Dy^2$. It is observe that $D = H_1 H_2 H_3 a$, where $H_1 = d > 0, H_2 = dc - b > 0, H_3 = bcd - a^2 d - b^2$. It is attach the system:

$$\dot{x} = y$$

$$\dot{y} = z$$

$$\dot{z} = u$$

$$\dot{u} = -ax - by - cz - du$$

here we choose to identify $\dot{V} = 2Dy^2$, so we have:

$$V = \frac{ac}{2}x^2 + adxy + \frac{c^2 - 2a}{2}y^2 + 2axz + cdyz + \frac{d^2 + c}{2}z^2 + u^2 + cyu + dzu + \frac{bd}{2}y^2 \quad (19)$$

$$\dot{V} = ady^2 - bcy^2 - 2byu - du^2 \quad (20)$$

For the nonlinear system we take the case when the term by is replaced by $\varphi(y)$, $\varphi(0) = 0$:

$$V = E + db \frac{y^2}{2} \quad (21)$$

$$V = E + d \int_0^y \varphi(y) dy \quad (22)$$

We consider the case of a system with two freedom degree under the matricial form:

$$M\ddot{X} + C\dot{X} + KX = 0 \quad (23)$$

where M is the mass matrix, C is the absorption matrix, K is the potential elastic matrix.

$$\begin{aligned} m_1\dot{x}_1c_{11}\dot{x}_1 + c_{12}\dot{x}_2 + k_{11}x_1 + k_{12}x_2 &= 0 \\ m_2\dot{x}_2 + c_{21}\dot{x}_1 + c_{22}x_2 + k_{21}x_1 + k_{22}x_2 &= 0 \end{aligned} \quad (24)$$

Passing at the system of fourth degree:

$$\dot{x} = u, \dot{y} = v \quad (25)$$

The characteristic polynomial $p = P(r)$ is:

$$P(r) = \begin{vmatrix} m_1r^2 + c_{11}r + k_{11} & c_{12}r + k_{12} \\ c_{21}r + k_{21} & m_2r^2 + c_{22}r + k_{22} \end{vmatrix} \quad (26)$$

By develop in series we obtain the polynomial of 4th degree for which is applying the theory above for find the function V . Another applications of this kind regarding the stability study of the dynamical system and their automatic regulation are in the papers [5], [6], [7], [8], [9].

REFERENCES

- [1] Dragos L., The principles of the analytical mechanics , Ed. Tech., Bucharest, 1976
- [2] Merkin D.R., Introduction in the movement stability theory, Moscova, Nauka, 1978
- [3] Stanescu N.D., Munteanu L., Chiroiu V., Pandrea N., Dynamical system Theory and Applications, Ed. Acad. Rom., Buc., 2007
- [4] Panovko Ya. G., Gubanova I.I., stability and oscilation in elastic systems, Ed. Moskva, 1964, (in russian)
- [5] Dumitrache I., The engineering of automatic regulation, Ed. Politehnica press, 2005, Bucharest
- [6] Chiriacescu S., Mechanical linear systems , Ed. Acad. Romane, Bucuresti, 2007.
- [7] Tanguy G.D., Fandloy L., Popescu D., Modelisation, identification et comandes des systems, Ed. Acad. Rom., 2004
- [8] Lupu M., Florea O. The study of the mass transfer in the case of the stationary movement of the viscous fluid between the cylindrical with parralel axis Proceedings of the International Conference, Acad. Forces Aeriene „Henri Coanda“, Brasov, 2007
- [9] Popescu D., Lupu. C, Pursuance systems in the industrial process Ed. Printech, 2004
- [10] Lupu M., Isaia F., The study of some nonlinear dynamical systems modelled by a more general Rayleigh - Van der Pole equation, J. Creative Mathematic and Inf., vol. 16/2007, pp 81-90