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## STOCHASTIC MODEL OF THE NONLINEAR FLUTTER WITH ONE DEGREE OF FREEDOM

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**Abstract:** This paper studies a flutter dynamic system type or car in case when  $F(t)$  is a random force. Graph of  $F(t)$  show the random value of road-wheel contact. The study of damping, aircraft dynamics efficiency and dynamics of car (for suspension) under random perturbation due to wind or current is made by stochastic methods.

**Keywords:** stochastic methods, random noise, flutter dynamic type system.

### 1. INTRODUCTION

The theory of the stochastic approximation is a challenging subject of the contemporary science. One the topics of high interest in this is the nonlinear flutter. The flutter phenomenon is to stabilize and absorb vibrations of a plane landing on a random field. On landing the aircraft thrust acts and due to wind resistance [1], [2], [5].

Flutter is a very important subject; it is in fact a violent oscillation of wings, that usually leads to catastrophic failure.

The same problem can put a jet landing on the water surface or a moving car. It is therefore necessary to study the conditions of stability or instability depending on random parameters.

In this paper we shall present an analysis of a nonlinear stochastic models for dynamic systems in connection with the flutter instability.

### 2. THE FLUTTER PHENOMENON

We consider a dynamical system which can rotate in the direction of wind, whose velocity is  $W$ . This simple type of classical flutter model is in fact a dynamic system with a single degree of freedom. The motion of system is restricted by an elastic spring of constant  $c$  and by a linear damping device of constant  $k$ . The system, of weight  $m$ , has a point fixed and makes the angle  $\theta$  with respect to the wind motion, this angle being in fact the angle of attack [4].

Under these conditions, we consider the following equation of motion of the system:

$$m\ddot{x} = F(t) + Q(x, \dot{x}), \quad Q(x, \dot{x}) = -cx - k\dot{x} \quad (1)$$

where  $Q = -cx - k\dot{x}$  forces are:

- $cx$  - elastic oscillator to minimize contact with the ground rigid;
- $k\dot{x}$  - dissipative viscous damper that dissipates (absorbs) oscillations;
- $F(t)$  - random disturbing force representing terrain.

Thrust consists of the result of the relative speed  $V_r$  and wind speed  $W$ . In the study of the critical speed occurs when wind speed is equal to the relative speed – to balance. Here we ask two questions:

1. the efficiency forces  $Q = -cx - k\dot{x}$  contributing to mobile when traveling over rough terrain with a button random. The study is done by stochastic methods and procedures.

2. the stability of dynamic system through mediation and center (statistical operation) was represented by the function  $F(t)$  of equation of motion. If we consider  $F(t)$  runway, it is replaced by the sine law started with a  $F(t) = H \sin \omega t$ .

The equation is written to a fixed reference point with vertical axis  $Ox$ :

$$m\ddot{a} = \overline{F}_e + \overline{F}_c + \overline{F}_p \quad (2)$$

where:

$\overline{F}_e$  – elastic force;

$\overline{F}_c$  – damping force;

$\overline{F}_p$  – disturbing force;

Further:  $|\overline{F}_e| = cx \Rightarrow \overline{F}_e = -cx$ ;

$$|\overline{F}_c| = k\dot{x} \Rightarrow \overline{F}_c = -k\dot{x}.$$

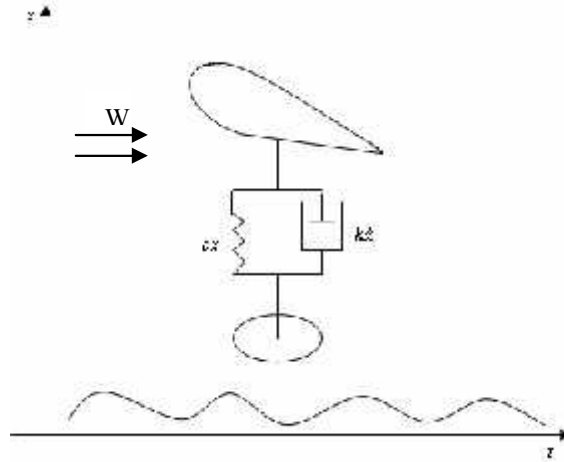


Figure 1: The dynamic system of flutter type

Dividing by  $m$ , the mass of system, equation (1) we get:

$$\ddot{x} + 2\gamma\dot{x} + \lambda^2x = \frac{H}{m} \sin \omega t \quad (3)$$

We noted the damping force  $2\gamma\dot{x}$ , with  $\lambda^2x$  elastic force and  $\frac{H}{m}F(t)$  random disturbance force multiplying. We believe that disruptive force, in probabilistic terms must be of the form:

$$F(t) = H \sin \omega t \quad (4)$$

Experimental data of spectral density  $F(t)$  is known as:

$$S_F(\omega) = \begin{cases} S_0 & 0 \leq \omega \leq \omega_0 \\ 0 & \omega > \omega_0 \end{cases} \quad (5)$$

Accordingly we have random dispersion interaction:

$$\sigma_{\ddot{x}}^2 = \int_0^\infty S_F(\omega) d\omega = \begin{cases} S_0\omega & 0 \leq \omega \leq \omega_0 \\ 0 & \omega > \omega_0 \end{cases} \quad (6)$$

In this case, frequency  $\omega_0$  is called frequency cut. Looking from statistically random function  $F(t)$ , we obtain that is a stationary random process with zero mean and variance given by (6), [6].

In solving equation (3) we seek the general solution as the sum of two individual roots [7]:

$$x = x_{\delta}^o + x_p^n \quad (7)$$

To do this, solve the equation feature:

$$\ddot{x} + 2\gamma\dot{x} + \lambda^2x = 0 \Rightarrow r^2 + 2\gamma r + \lambda^2 = 0 \quad (8)$$

hence the roots:

$$r_{1,2} = -\gamma \pm \sqrt{\gamma^2 - \lambda^2} \quad (9)$$

but  $\gamma < \lambda$  because  $F_c < F_e$  dissipative force is less than the elastic. Therefore,  $r_{1,2} = -\gamma \pm i\delta$ , noted by  $\delta^2 = \gamma^2 - \lambda^2$ .

These roots lead us to two particular solutions of equation (8). In these conditions we get:

$$x_{\delta}^o = e^{-\gamma t} (c_1 \cos \delta t + c_2 \sin \delta t) \quad (10)$$

The term  $x_{\delta}^o$  dissipates (stabilizes) quickly because  $\lim_{t \rightarrow \infty} e^{-\gamma t} = 0$  and most important will be the solution that influence particular disturbance.

If  $x_p^n = M \sin \omega t + N \cos \omega t$  determine M, N introducing in homogeneous equation by identifying

$$x_p^n = M \left\{ \sin \omega t + \frac{N}{M} \cos \omega t \right\} = \frac{M}{\cos \theta} (\sin \omega t \cdot \cos \theta - \cos \omega t \cdot \sin \theta), \quad \frac{N}{M} = \operatorname{tg} \varphi = -\operatorname{tg} \theta$$

After calculations we find:

$$x_p^n = \frac{M}{m \sqrt{(\lambda^2 - \omega^2)^2 + 4\gamma^2 \omega^2}} \sin(\omega t - \theta) \quad (11)$$

Returning to the solution of the equation  $x = x_g^0 + x_p^n$ ,  $x_g^0$  is canceled and  $x = x_p^n$ .

Comments:

- $x_p^n$  can and Laplace transform with zero conditions;
- for  $\gamma$  small ( $\gamma \rightarrow 0$ ), if  $\lambda^2 \rightarrow \omega^2$  we have resonance phenomenon that increases the amplitude  $\ddot{x} + \omega^2 x = \frac{N}{m} \sin \omega t$ . In this case  $x_p^n = Mt \sin \omega t$  and amplitude increase as  $Mt \rightarrow \infty$  because the elastic shock  $cx$  resonate with the way the equation  $\varepsilon \sin \omega t$  [3].

### 3. STOCHASTIC MODEL OF THE NONLINEAR FLUTTER

As show in Fig. 1, the structural dynamics of the system is described by a single equation of force equilibrium:

$$m\ddot{x} + c\dot{x} + kx = \frac{1}{2}\rho W^2 SC(\theta) \quad (12)$$

As the lift force is  $L = \frac{1}{2}\rho W^2 SC(\theta)$ ;  $\rho$  is the density of fluid,  $S$  is the surface of the plate exposed to the fluid flow and  $C(\theta)$  is the lift coefficient. (fig. 1).

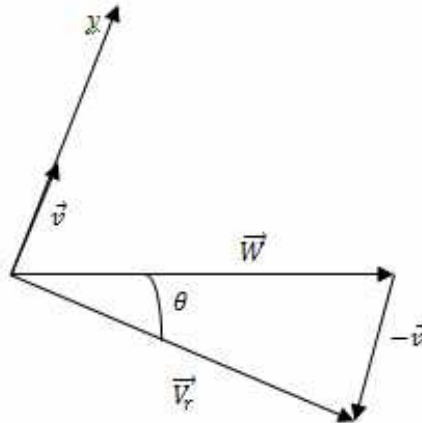


Figure 2 Dynamic system structure

According to fig. 2 the relative velocity has the magnitude  $V_r = \sqrt{\dot{x}^2 + W^2}$  and makes angle with the axis  $Ox$  given by  $\operatorname{tg} \theta = \frac{\dot{x}}{W}$ . The aerodynamic coefficient  $C(\theta) = C(\dot{x})$  may be described with a good approximation, by the following polynomial representation

$$C(\theta) = a_1 \theta - a_2 \theta^2 + a_3 \theta^3 - a_4 \theta^7 + \dots \quad (13)$$

Indicated by analytical and experimental. For small angles of attack approximating  $\operatorname{tg} \theta \approx \sin \theta \approx \theta$  we may use for low frequencies a quasisteady representation of the lift coefficient. For example,

$$\frac{1}{2}\rho W^2 SC(\theta) = \frac{1}{2}\rho W^2 S \left[ a_1 \frac{\dot{x}}{W} - a_2 \left( \frac{\dot{x}}{W} \right)^2 + a_3 \left( \frac{\dot{x}}{W} \right)^3 - a_4 \left( \frac{\dot{x}}{W} \right)^7 + \dots \right] \quad (14)$$

Passing into the phase space  $(x, y)$  where  $y = \dot{x}$  and using (1) and (3), we obtain the following system of 1st order ODE:

$$\begin{cases} \dot{x} = y \\ \dot{y} = -\frac{kx}{m} + \frac{1}{m} \left( \frac{1}{2}\rho W^2 S a_1 - C \right) y - \frac{1}{2}\rho \frac{S a_2}{mW} y^2 + \frac{1}{2}\rho \frac{S a_3}{mW} y^3 \dots \end{cases} \quad (15)$$

In the case of equilibrium in the first approximation, we have the solution  $y = 0$ . Using the expression of the critical velocity  $V_c = \frac{2C}{\rho S a_1}$  and introducing the notations:  $a = \frac{W}{V_c}$ ,  $b = \frac{K}{m}$  the system (15) becomes:

$$\begin{cases} \dot{x} = y = F_1(x, y) \\ \dot{y} = -bx + y[-1 + a[1 - f(y)]] = F_2(x, y) \end{cases} \quad (16)$$

Where

$$f(y) = \frac{A_2}{A_1} y^2 - \frac{A_3}{A_1} y^4 + \frac{A_4}{A_1} y^7 \text{ și } V_c = \frac{2C}{\rho S A_1} \quad (17)$$

The equilibrium critical point of the non-linear system, for  $F_1 = 0$  și  $F_2 = 0$  becomes  $O^*(x^* = 0, y^* = 0)$ . We will deal with the stability around this point and, due to the complexity of the non-linear system, we will use various criteria such as the stability in first approximation.

#### 4. STABILITY IN FIRST APROXIMATION

The Jacobian of the system (16) is:

$$J = \begin{pmatrix} \frac{\partial F_1}{\partial x} & \frac{\partial F_1}{\partial y} \\ \frac{\partial F_2}{\partial x} & \frac{\partial F_2}{\partial y} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -b & -1 + a[1 - f(y) - yf'(y)] \end{pmatrix} \quad (18)$$

From (16) we obtain the linear system:

$$\begin{cases} \dot{x} = y \\ \dot{y} = -bx + y(-1 + a) \end{cases} \quad (19)$$

where:

$$J(P^*) = \begin{pmatrix} 0 & 1 \\ -b & a-1 \end{pmatrix} \quad (20)$$

Denoting by  $P(r)$  the characteristic polynomial, the equation  $P(r) = 0$  becomes:

$$P(r) = \det(J - rI) = r^2 + (1 - a)r + b = 0 \quad (21)$$

The study reduces to compare the wind speed  $W$  with  $V_C$ , i.e.:

- if  $0 < a < 1 \Leftrightarrow W < V_C$ , where  $a = \frac{W}{V_C}$  the linear system (19) is asymptotic stable about the point  $O^*$  and the non-linear system (16) is asymptotic stabil.
- if  $a > 1$  the linear system (19) is not stable, implying thus that for  $W > W_{cr}$  the system (16) is not stabil.
- if  $a = 1$ ,  $W_{cr} = V_C$  we have simple stability for the system (19) and one cannot decide on the stability of the non-linear system (16).

Analyzing the sign of the expression  $\Delta = (1 - a)^2 - 4b$  in the plane  $(a, b)$  we can characterize the stability of the traiectories:

1. if  $\Delta > 0$ ,  $D_1 = \{(a, b) \mid 0 < a < 1, b < \frac{(1-a)^2}{4}\}$  with the initial condition  $t = 0, P_0(x_0, y_0)$  the traiectories of the system (16),  $x = x(t), y = y(t)$  are curves which tend asymptotically to the stable point  $O^*$ .
2. if  $\Delta < 0$ ,  $D_2 = \{(a, b) \mid 0 < a < 1, b > \frac{(1-a)^2}{4}\}$  then the traiectories of the system (16),  $x = x(t, x_0, y_0), y = y(t, x_0, y_0)$  are spirals tending asymptotic to the point  $O^*$ .
3. for  $a > 1$  we have instability.
4. the critical case  $W = V_{cr}, a = 1, \Delta = 0$  (on the parabola P) the reduced linear system  $\dot{x} = y, \dot{y} = -bx$  has traiectories tending to an ellipse (E):

$$E: \begin{cases} x = x_0 \cos \sqrt{bt} + \frac{y_0}{\sqrt{b}} \cos \sqrt{bt}, \\ y = -x_0 \sin \sqrt{bt} + y_0 \cos \sqrt{bt} \end{cases} \quad (22)$$

Which are simple stable concerning the center  $O^*$ ; nothing can be said about the stability of the system (16).

#### 5. CONCLUSION

To this end, we mention that our previous analysis may be extended by taking into account the damping of the system, which will influence its state of stability/instability, but this task is left for a future study.

#### 6. REFERENCES

- [1] Buzdugan Gh., Fetcu L., Radeş M., Vibrații mecanice, Editura Didactică și Pedagogică, București, 1982.
- [2] Chiriacescu S., Sisteme mecanice liniare: o introducere, Editura Academiei Române, București, 2007.
- [3] Jaeger, J., Newstead G. Introducere în teoria transformării Laplace, Editura Tehnică, București, 1971.
- [4] Lupu M.: Mathematical Models of the Mechanics of Continuum Media, Transilvania University Publishing House, Brasov, 1985.
- [5] Orman, G. V., Handbook of limit theorems and stochastic approximations, Transilvania University Press, Brasov, 2003.
- [6] Orman, G. V., Măsură și procese aleatoare, Editura Universității Transilvania din Brașov, 2000.
- [7] Marinescu C., Marin M., Ecuatii diferențiale și integrale. Editura Tehnică, București, 1996.